

Amendments to the Specification:

1. Please replace paragraph 3, page 7 beginning with "Figure 2A is" with the following paragraph:

Figure 2[[A]] is a top view of the transfer pump of Figure 1;

2. Please replace paragraph 1, page 12 beginning with "Referring again" with the following paragraph:

Referring again to Figure 8, certain features of a preferred impeller of the pump 100 can be seen. Preferred impeller 210 comprises a base flange 212 upon which are formed a plurality of vanes, e.g. vanes 214 and 216. Vanes 214 and 216 (and others if present) each comprise an outer face, e.g. faces 215 and 217, respectively. Faces 215 and 217 are formed with a precision flat surface that is substantially perpendicular to the axis of rotation of impeller 210, such rotation being indicated by arrow 299. With such a dimensional arrangement, the axial runout of faces 215 and 217 (and others if present) is minimized. Faces 215, 217, and others present are coplanar, such that faces 215, 217, etc. define a plane or overall vane face that is perpendicular to the axis of rotation of impeller 210. In general, it is preferable that impeller faces 215/217 have an axial runout of less than about 0.006 inch, and more preferably less than about 0.003 inch. Such minimal runout and coplanarity of faces 215, 217, etc. are important in providing a pump of high efficiency, as will be explained presently.

3. Please replace paragraph 2 on page 22 beginning with "In the preferred embodiment" with the following paragraph:

In the preferred embodiment, fan 330 is an axial-type fan, having vanes optimally formed to efficiently draw air along the axis thereof and directs it radially. Figure 13 is an axial cross sectional view of pump that depicts housing section 120 and cooling fan 330 disposed therein. Referring to Figure 13 and to Figure 19, it can be seen that at this axial location along housing portion 120, housing wall 123 is formed into an overall volute cavity 121 within which is disposed fan 330. Volute cavity 121 preferably has a double volute shape for the effective discharge of air from within housing portion 120 by fan 330. Housing portion 120 comprises an upper volute 130 and a lower volute 131.

4. Please replace paragraph 1, page 13 beginning with "operably joined" with the following paragraph:

operably joined to motor shaft 312 by nut 314. The motor (not shown) of pump 100 is mounted in housing 120 (see Figure 1) with the central axis 399 of motor shaft 312 therein being precisely aligned such that the plane defined by impeller faces 215, 217, etc. is substantially parallel to face 418 of cover 410. The running clearance 498 499—between face 418 and faces 215/217 is preferably between about 0.01 and about 0.04 inch, and more preferably between about 0.01 and about 0.02 inch. Faces 215/217 having minimal runout as described above enable such small running clearances, and thereby enable high pump efficiency, since very little liquid can pass through the running

clearance, and thus substantially all of the liquid is accelerated by the vanes of the impeller.

5. Please replace paragraph 3, page 16 beginning with "Referring to" with the following paragraph:

Referring to Figures 9 and 11, the initial spinning of impeller 210 ejects an initial surge of liquid out of outlet port 154, with liquid level in cavity casing-156 falling from level 194 (see Figure 10) to about level 193 , whereupon pump 100 achieves a period of pseudo-steady state operation during self-priming. During this period, the liquid level 193 in cavity casing-156 is maintained relatively constant. However, self priming of the pump occurs due to the recirculation of liquid from pump volute 169 to cavity casing-156, and back into pump volute 169, and so forth.

6. Please replace paragraph 2, page 26 beginning with "Referring again" with the following paragraph:

Referring again to Figures 17A, 17B and 19, cover 450 preferably comprises protective upper rib 478 and protective lower rib 479, for protection of switch 370 installed in hole 470. Cover 450 further comprises left ear 480 and right ear 481 having right threaded bore 482 and left threaded bore (not shown). Cover 450 further comprises rectangular slots 484 and 485 for housing the motor brushes (not shown) and gusseted extensions 486 and 487, in which are provided holes 488 and 489. Stator 350 of motor 310 is fastened to cover 450 by

engagement of threaded fasteners 495 and 496 (Figure 19) (not shown) with holes 488 and 489. Referring to Figure 18, caps 490 are engaged in left threaded bore 482 and right bore (not shown) to seal and hold the motor brushes 361 and 362 therein. In a further embodiment, rectangular slots

7. Please replace paragraph 1, page 11 beginning with "Referring again" with the following paragraph:

Referring again to Figure 5, fluid is pumped by pump 100 along the flow path 199 within pump casing 150, and is discharged from outlet port 154 in pump casing 150. Referring to Figure 6, flow path 199 within pump casing 150 is depicted in more detail. Fluid is first accelerated from axial flow into the eye of impeller 210 by the spinning action 299 (see Figure 8) thereof, resulting in a generally cycloidal trajectory 198, wherein the fluid is discharged tangentially from impeller 210, and flows out from exit flare 162 of volute chamber 160 and volute 169 into main cavity 156 of pump casing 150. The fluid then flows generally along path 197 within main cavity 156, and then along path 196 within outlet passageway 145158, whereupon the fluid is discharged at outlet 154.